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A further discussion on the effective thermal conductivity of metal foam: an improved model

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Abstract

In this study, we explain the causes and effects of the geometrical impossible result encountered in the widely adopted tetrakaidecahedron model (Boomsma and Poulikakos, 2001; Dai *et al.*, 2010) for the effective thermal conductivities (ETCs) of metal foam. The geometrical impossible result is successfully eliminated by accounting for the size variation of the node with porosity. The improved model provides predictions of ETCs that are more precise than available models. For aluminum foams ($k_s = 218 \text{ W m}^{-1}\text{K}^{-1}$) using water and air as fluid media, the relative root-mean-square (RMS) deviation of the present predictions from the experimental data is about 5.3%; for the reticulated vitreous carbon (RVC) foams ($k_s = 8.5 \text{ W m}^{-1}\text{K}^{-1}$), the relative RMS deviation is about 7.4%.

Key words: effective thermal conductivity; foam structure; node size; deviation; improved model.

Nomenclature

a	foam ligament radius (m)	L	ligament length (m)
d	dimensionless foam ligament radius	R	thermal resistance ($\text{m}^2 \text{ K W}^{-1}$)
e	dimensionless cubic node length	r	cubic node length (m)
f	function	V	volume (m^3)
k	thermal conductivity ($\text{W m}^{-1} \text{ K}^{-1}$)		

Subscripts

A	unit cell layer	D	unit cell layer
B	unit cell layer	eff	effective
C	unit cell layer	f	fluid

i A, B, C, D s solid

21 *Greek symbol*

ε porosity

22 **1. Introduction**

23 High-porosity metal foams are promising materials for thermal management applications.
24 Since the effective thermal conductivity (ETC) is one of their most important thermal properties,
25 an accurate evaluation of it becomes especially important. Studies on modeling the ETC of metal
26 foams have been carried out numerically [1-3], experimentally [4-6] and analytically [5-12].
27 Among these approaches, the analytical approaches are less time consuming but more universal,
28 and have attracted the attention of investigators. A review of the analytical approaches for
29 prediction of the ETC has been conducted by Coquard and Baillis [13, 14], and Randrianalisoa
30 and Baillis [15].

31 One of the most widely used analytical approaches was developed by Boomsma and
32 Poulikakos [10] who first used the idealized three dimensional tetrakaidecahedron model to
33 predict the metal foam ETC. Predictions were reported to accurately match the experimental data.
34 However, Dai *et al.* [11] pointed out a few problems in their work. Dai *et al.* [11] extended the
35 model by accounting for the ligament orientation. Predictions of the extended model were
36 compared with the experimental data [5], and a relative RMS deviation of about 12% was
37 observed. The deviation indicated that there was still room for improvement. In addition, results
38 obtained in Ref. [11] showed that, as the porosity decreased, the diameter of the ligament became
39 longer than the length of the node, which led to a geometrical impossible result. The diameter
40 of the ligament should be shorter than the length of the node (see Fig. 1), which was a basic
41 assumption in the development of the model.

42 In this paper, the tetrakaidecahedron model originally proposed by Boomsma and Poulikakos
43 [10] and later extended by Dai *et al.* [11] is first discussed. The causes and effects of the
44 geometrical impossible results are examined and explained. The model is further improved by
45 accounting for the size variation of the nodes. We then show that the geometrical impossible
46 results are eliminated. Lastly, predictions of our improved model are compared with several other
47 analytical solutions as well as experimental data available in literature. It is shown that the current
48 model has a steadily high precision in predicting the ETC of high porosity foams with a wide
49 range of phase conductivity ratios (k_s / k_f).

50 **2. Calculation of the Effective Thermal Conductivity**

51 It is important to note that, for a better understanding of the present discussion, reader should
52 be familiar with the analytical approaches developed by Boomsma and Poulikakos [10], and Dai
53 *et al.* [11]. Therefore, in this part of the discussion, we give a brief review of how the ETC is
54 calculated using their approaches. For more detailed discussions, reader may refer to Refs. [10,
55 11].

56 **2.1. The Unit Cell of the Tetrakaidecahedron Model**

57 Cubic nodes and cylindrical ligaments were used to represent the actual components of the
58 foam network, as is shown in Fig. 1(a). As the lump shape at the ligament intersection varies with
59 the foam porosity, the simplified spherical geometry is adopted for various porosities based on the
60 fact that the lump volume has more significant effect on ETC than its shape does. The length of
61 the node is r , the radius of the ligament is a and its length is L (from node center to node
62 center). Based on the symmetry of the idealized model and one-dimension heat conduction along
63 the z axis, a representative unit cell which contains all geometrical characteristics of the

64 tetrakaidecahedron model was selected. The height of the unit cell in the z direction is
 65 $L\sqrt{2}/2$. While the length of the other two sides in the $x-y$ plane are both $L\sqrt{2}$. It can be
 66 proved that the ETC of the selected unit cell is equal to that of the tetrakaidecahedron model.

67 2.2. Effective Thermal Conductivity

68 In order to calculate the ETC, the unit cell is divided into four distinctive vertical layers along
 69 the z axis, namely A, B, C and D, as is shown in Fig. 1(b). The heights of the four layers are:

$$70 L_A = a, L_B = r/2 - a, L_C = L\sqrt{2}/2 - r \text{ and } L_D = r/2 .$$

71 According to the extend model (accounting for the ligament orientation) proposed by Dai *et al.*
 72 [11], the thermal resistance of each layer is

$$73 R_A = \frac{4dL}{[2e^2 + \pi d(1-e)]k_s + \{4 - [2e^2 + \pi d(1-e)]\}k_f} \quad (1a)$$

$$74 R_B = \frac{(e-2d)L}{e^2k_s + (2-e^2)k_f} \quad (1b)$$

$$75 R_C = \frac{2(\sqrt{2}-2e)L}{\pi d^2k_s\sqrt{2} + 2(2-\pi d^2\sqrt{2})k_f} \quad (1c)$$

$$76 R_D = \frac{2eL}{e^2k_s + (4-e^2)k_f} \quad (1d)$$

77 where k_s is the thermal conductivity of the solid and k_f is thermal conductivities of the fluid,
 78 d and e are non-dimensional parameters, defined as: $d \equiv a/L$ and $e \equiv r/L$.

79 The overall thermal conductivity is calculated by assuming that the thermal resistances of the
 80 layers are connected in series. Based on the Fourier law of heat conduction, the ETC can be
 81 written as

$$82 k_{\text{eff}} = \frac{L_A + L_B + L_C + L_D}{R_A + R_B + R_C + R_D} \quad (2)$$

83 Eq. (2) can be written as

$$84 k_{\text{eff}} = f_1(d, e) \quad (3)$$

85 here, f_1 is a known function.

86 The porosity ε , which is defined as the ratio of the solid volume to the total volume can be

87 calculate based on d and e as

$$88 \quad \varepsilon = 1 - \frac{\sqrt{2}}{2} [de^2 + \frac{\pi d^2}{2}(1-e) + (\frac{e}{2} - d)e^2 + \pi d^2(1-e\sqrt{2}) + \frac{e^3}{4}] \quad (4)$$

89 Solving for d in Eq. (4) gives

$$90 \quad d = \left[\frac{\sqrt{2}(2 - 2\varepsilon - \frac{3\sqrt{2}}{4}e^3)}{\pi(3 - e - 2e\sqrt{2})} \right]^{\frac{1}{2}} \quad (5)$$

91 Substituting Eq. (5) into Eq. (3) gives

$$92 \quad k_{\text{eff}} = f_2(\varepsilon, e) \quad (6)$$

93 where f_2 is another known function. Here, once the value of e is given, the ETC can be

94 calculated purely by porosity.

95 **3. Improved Model**

96 We present a discussion on the model of Dai *et al.* [11]; highlighting the possible area of

97 improvement. We then discuss the reason for the appearance of the geometrical impossible result.

98 Our proposed model is then presented.

99 **3.1. Revisit Dai *et al.*'s Model**

100 Precision: In order to use Eq. (6) to predict the ETC, the value of e should first be calibrated.

101 According to Ref. [11], a value of $e = 0.198$ was found to minimize the relative RMS deviation

102 of predictions from the experimental data [5]. This deviation is about 12%, which indicates that

103 there is still room for improvement.

104 The Geometrical Impossible Result: It should be true that $r/a > 2$ to ensure that the length of

105 the node is larger than the diameter of the ligament. However, as has been mentioned in Ref. [11],

106 this requirement can hardly be fulfilled with $e = 0.198$. The reason for the appearance of this

107 geometrical impossible result is explained next.

108 **3.2. Causes and Effects of the geometrical impossible results**

109 According to Eq. (4) or the structure of the tetrakaidecahedron model (see Figs. 1(a), 1(b)), a
110 decrease in porosity ε , is attributed to an increase in d when e is held constant and vice-versa.
111 Here, d and e can be considered to represent the diameter of the ligament and the length of the
112 node. In the model of Boomsma and Poulikakos [10] and Dai *et al.* [11], the parameter e was set
113 as a constant value. Therefore, according to Eq. (4), a decrease in porosity can only be realized by
114 an increase in the diameter of the ligament d . As a result, as the porosity decreases, the diameter of
115 the ligament increases while the length of the node remains constant, and eventually, the former
116 exceeds the latter, leading to geometrical impossible results. In fact, the smaller the value of e ,
117 the more likely the geometrical impossible result occurs. As a result, geometrical impossible
118 results were encountered more frequently in Ref. [11] than in Ref. [10].

119 **3.3. Current Model**

120 We improve on the model of Dai *et al.* [10] and eliminate the geometrical impossible result
121 by accounting for the changing foam structure with porosity through the variable e . Since the
122 experimental data contains information of the foam structure, using the experimental data, we can
123 find how e varies with porosity. For a given porosity, we calibrate the e value by comparing
124 the predictions made by Eq. (6) against experimental data (Ref. [5]). As a result, values of e for
125 ten given porosities are obtained, as is shown in Fig. 2. The parameter e can be fitted by a third
126 order polynomial function of the porosity ε (Fig. 2) as

$$127 \quad e = a + b\varepsilon + c\varepsilon^2 + d\varepsilon^3 \quad (7)$$

128 where $a = 327.25811$, $b = -1075.55645$, $c = 1182.83207$ and $d = -434.55535$. The fitting error is

129 less than 1%.

130 **4. Model Validation**

131 After obtaining the function between e and porosity, the ETC can be predicted as a function of
132 porosity. Substituting Eq. (7) into Eq. (6), gives:

$$133 \quad k_{\text{eff}} = f(\varepsilon) \quad (8)$$

134 where f is a known function.

135 When Eq. (8) is used to compute the ETCs for the experimental data in Ref. [5], the relative
136 RMS deviation is about 5.0% for water-saturated foams, and 5.6% for air-saturated foams. The
137 ratios of the node length to the ligament radius are also predicted using the current approach. As a
138 result, within the porosity of 0.905 to 0.978, the r/a ratios are always > 2.0 (decreasing from 6.33
139 to 2.71). Thus, the geometrical impossible results are eliminated.

140 Figure 3 shows comparisons of our model with selected models. As a result of the changing e
141 values with porosity, our model is capable of capturing the non-linear variations in the ETCs as
142 function of porosity. The relative RMS deviations from water and air experimental data of the
143 present, Yang's, Dai's and Paek's predictions are 5.3%, 11.7%, 13.2% and 13.3% respectively.
144 Since the e value is calibrated from the experimental data in Ref. [5], the high precision against
145 the experimental data in Ref. [5] is expected. To validate our model, we use Eq. (8) to compute the
146 ETCs for the experimental data reported by Phanikumar and Mahajan [16]. They reported the
147 ETCs of air-saturated Al foams with porosity ranging from 0.899 to 0.959. Due to a similar foam
148 geometry, the present model is capable of accurately predicting the variation trend of the ETCs
149 with porosity; with a relative RMS deviation of about 12.1%.

150 To further assess the validity of our improved model as well as the fitted e value (Eq. 7).

151 We use our model to predict the ETCs of high porosity RVC foams (which have similar structure
152 with high porosity metal foams), and the results are compared with experimental measurements
153 reported in Ref. [6]. As a result, for water-saturated RVC foams, all the analytical models can
154 accurately predict the ETCs (RMS deviation < 10%). When it comes to air-saturated RVC foams,
155 Yang's model and Peak's model are less accurate with a RMS deviation of more than 21.8%. The
156 relative RMS deviations of the present model are relative small; 7.2% and 7.6% for
157 water-saturated and air-saturated RVC foams respectively. These results indicate that our
158 improved model have a wider range of applicability.

159 **5. Conclusions**

160 We accounted for the size variation of the node with porosity and successfully eliminate the
161 geometrical impossible results. The improved model provides more accurate predictions of the
162 ETCs. Comparisons with other analytical models as well as experimental data validates that our
163 model has a steadily high precision in predicting ETCs of foams with a wide range of solid phase
164 to liquid phase conductivity ratios (k_s / k_f).

165 **Conflict of interest**

166 None.

167 **Acknowledgments**

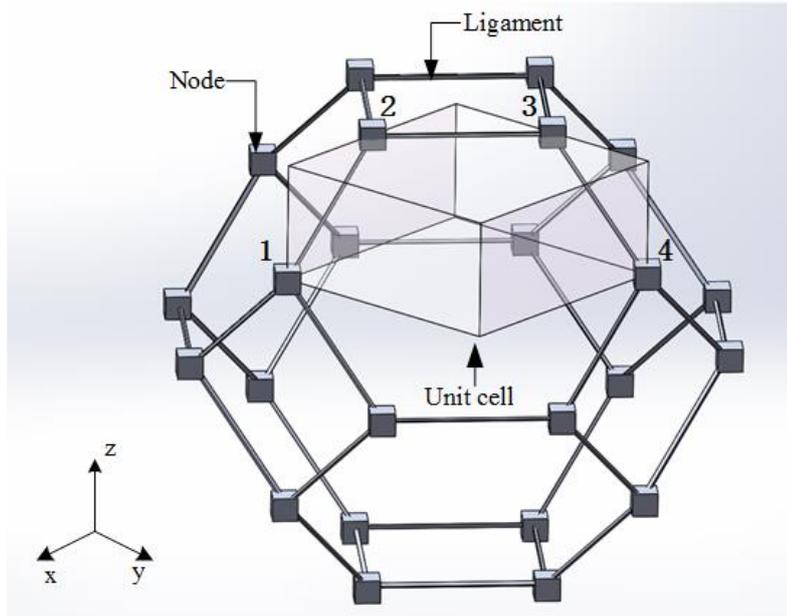
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171 **References**

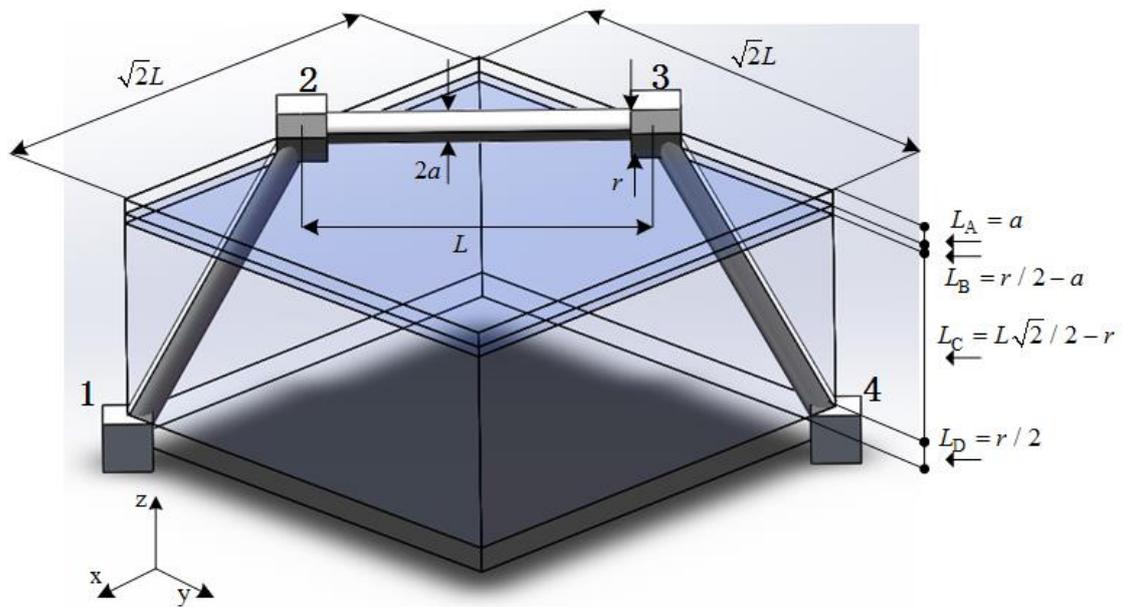
172 [1] M.A. Mendes, S. Ray, D. Trimis, An improved model for the effective thermal conductivity of
173 open-cell porous foams, *International Journal of Heat and Mass Transfer* 75 (2014) 224-230.

- 174 [2] M.A. Mendes, S. Ray, D. Trimis, A simple and efficient method for the evaluation of effective
175 thermal conductivity of open-cell foam-like structures, *International Journal of Heat and Mass*
176 *Transfer* 66 (2013) 412-422.
- 177 [3] M. Wang, N. Pan, Modeling and prediction of the effective thermal conductivity of random
178 open-cell porous foams, *International Journal of Heat and Mass Transfer* 51 (5) (2008) 1325-1331.
- 179 [4] N. Dukhan, K.C. Chen, Heat transfer measurements in metal foam subjected to constant heat
180 flux, *Experimental Thermal and Fluid Science* 32 (2) (2007) 624-631.
- 181 [5] V.V. Calmidi, R.L. Mahajan, The effective thermal conductivity of high porosity fibrous metal
182 foams, *Journal of Heat Transfer* 121 (2) (1999) 466-471.
- 183 [6] A. Bhattacharya, V.V. Calmidi, R.L. Mahajan, Thermophysical properties of high porosity
184 metal foams, *International Journal of Heat and Mass Transfer* 45 (5) (2002) 1017-1031.
- 185 [7] J.W. Paek, B.H. Kang, S.Y. Kim, J.M. Hyun, Effective Thermal Conductivity and Permeability
186 of Aluminum Foam Materials, *International Journal of Thermophysics* 21 (2) (2000) 453-464.
- 187 [8] X. Fu, R. Viskanta, J.P. Gore, Prediction of effective thermal conductivity of cellular
188 ceramics, *International Communications in Heat and Mass Transfer* 25 (2) (1998) 151-160.
- 189 [9] C.Y. Wang, C. Beckermann, A two-phase mixture model of liquid-gas flow and heat transfer in
190 capillary porous media-I. Formulation, *International Journal of Heat and Mass Transfer* 36 (1993)
191 2747-2747.
- 192 [10] K. Boomsma, D. Poulikakos, On the effective thermal conductivity of a three-dimensionally
193 structured fluid-saturated metal foam, *International Journal of Heat and Mass Transfer* 44 (4)
194 (2001) 827-836.
- 195 [11] Z. Dai, K. Nawaz, Y.G. Park, J. Bock, A.M. Jacobi, Correcting and extending the
196 Boomsma-Poulikakos effective thermal conductivity model for three-dimensional, fluid-saturated
197 metal foams, *International Communications in Heat and Mass Transfer* 37 (6) (2010) 575-580.
- 198 [12] X.H. Yang, J.X. Bai, H.B. Yan, J.J. Kuang, T.J. Lu, T. Kim, An analytical unit cell model for
199 the effective thermal conductivity of high porosity open-cell metal foams, *Transport in Porous*
200 *Media* 102 (3) (2014) 403-426.
- 201 [13] R. Coquard, D. Baillis, Numerical investigation of conductive heat transfer in high-porosity
202 foams, *Acta Materialia* 57 (18) (2009) 5466-5479.
- 203 [14] R. Coquard, D. Rochais, D. Baillis, Conductive and radiative heat transfer in ceramic and
204 metal foams at fire temperatures, *Fire Technology* 48 (3) (2012) 699-732.
- 205 [15] J. Randrianalisoa, D. Baillis, Thermal conductive and radiative properties of solid foams:
206 Traditional and recent advanced modelling approaches, *Comptes Rendus Physique*, 2014, in press.
- 207 [16] M.S. Phanikumar, R. L. Mahajan, Non-Darcy natural convection in high porosity metal
208 foams, *International Journal of Heat and Mass Transfer* 45 (18) (2002) 3781-3793.

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(a)



(b)

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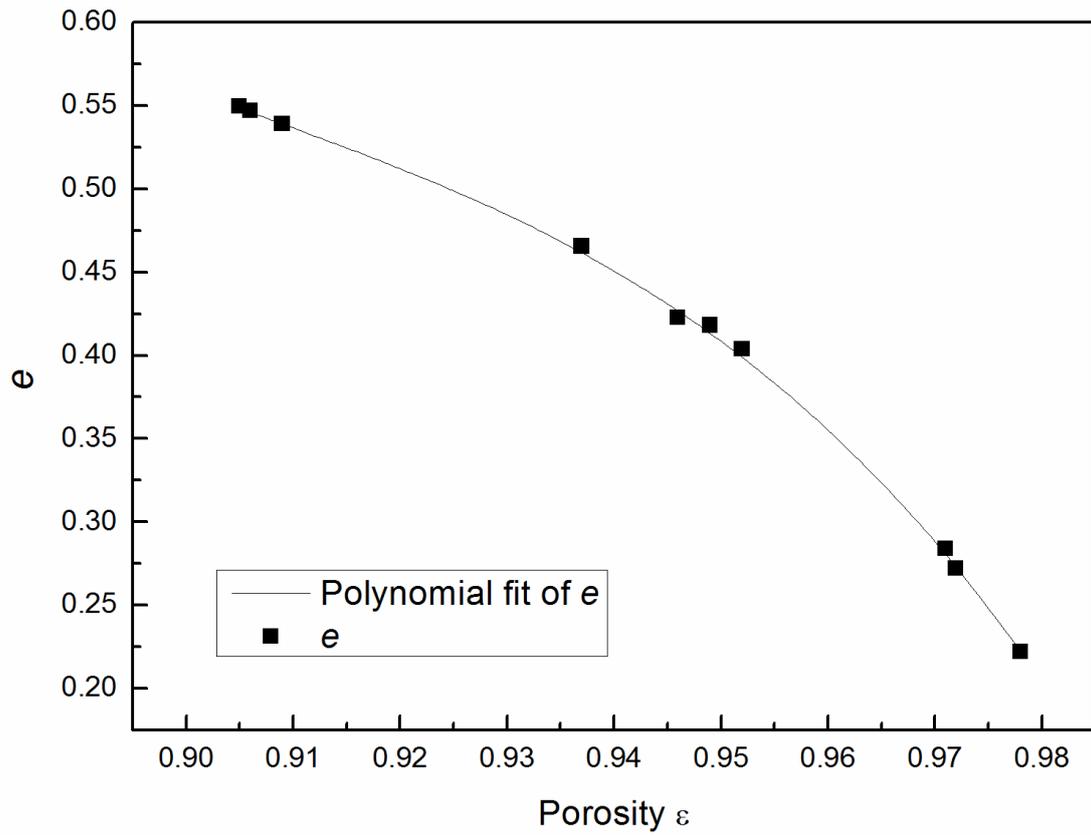
Fig. 1 (a) The tetrakaidecahedron model and (b) four distinctive layers for the unit cell

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Fig. 2 Calibrated e values

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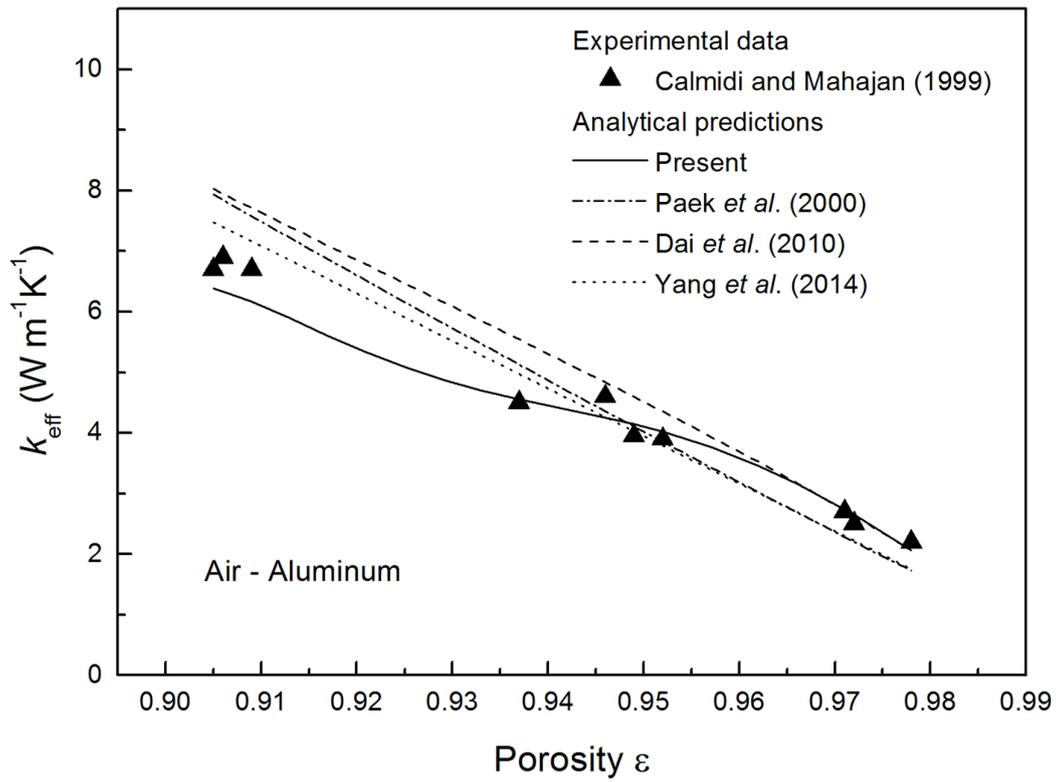
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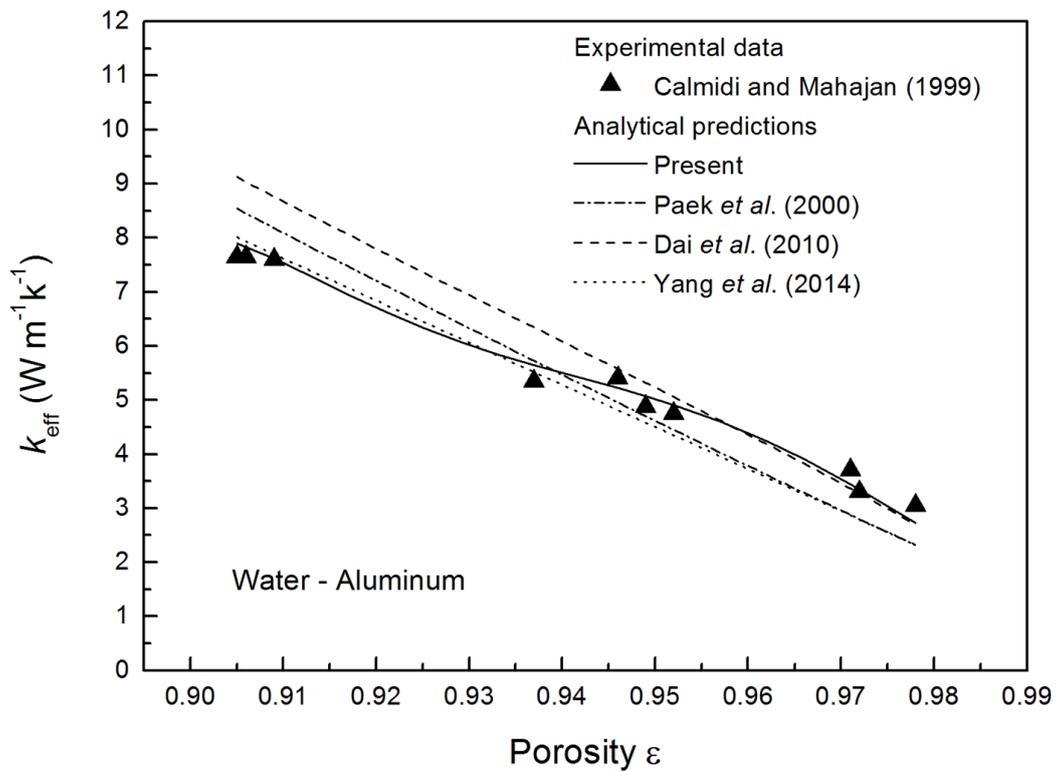
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(a)



(a)

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232 Fig. 3 Comparisons between the analytical models and the experimental data of (a) air-aluminum
 233 and (b) water-aluminum