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# Research in the development of finite element software for creep damage analysis

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**Abstract:** This paper reports the development of finite element software for creep damage analysis. Creep damage deformation and failure of high temperature structure is a serious problem for power generation and it is even more technically demanding under the current increasing demand of power and economic and sustainability pressure. This paper primarily consists of three parts: 1) the need and the justification of the development of in-house software; 2) the techniques in developing such software for creep damage analysis; 3) thirdly, the validation of the finite element software conducted under plane stress, plane strain, axisymmetric, and 3 dimensional cases. This paper contributes to the computational creep damage mechanics in general.

**Key words:** finite element software, creep damage, CDM, axisymmetric, validation.

## 1. Introduction

Creep damage mechanics has been developed and applied to analyze creep deformation and the simulation for the creep damage evolution and rupture of high temperature components [1].

The computational capability can only be obtained by either the development and the application of special material subroutine in junction with standard commercial software (such as ABAQUS or ANSYS) or the development and application of dedicated in-house software.

The need of such computational capability and the justification for developing in-house software was identified and which was reported in the early stage of this research [2, 3]. Essentially, the creep damage problem is of time dependent, non-linear material behavior, and multi-material zones. Hyde [4] and Hayhurst [5] have reported the development and the use of their in-house software for creep damage

analysis; furthermore, Ling et al [6] has presented a detailed discussion and the use of Runge-Kutta type integration algorithm. On the other hand, it was noted that Xu [7] revealed the deficiency of Kachanov-Rabatnov-Hayhurst (KRH) formulation and proposed a new formulation for the multi-axial generalization in the development of creep damage constitutive equations. The new creep damage constitutive equations for low Cr-Mo steel and for high Cr-Mo steel are under development in this research group [8, 9].

The purpose of this paper is to present the finite element method based on Continuum Damage Mechanics (CDM) to develop FE software for creep damage mechanics. More specifically, it summarizes the current state of how to obtain such computational capability then it concludes with a preference of in-house software; secondly, it reports the development of such software including the development of finite element algorithms based on CDM for creep damage analysis, and a flow diagram of the structure of new finite element software has

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been completed to be guided in developing in-house FE software, and the use of some standard subroutines in programming; thirdly, the development and the validation of the finite element software conducted so far include plane stress, plane strain, axisymmetric case, and 3D case were reported.

## 2. Current Finite Element Software for Creep Damage Analysis

### 2.1 The industrial standard FE package

The current industrial standard FE package is not able to provide the creep damage analysis capability and it can be expanded with the development and use of special subroutine.

**Table 1 The main industrial standard FE package.**

Industrial standard FE package	Samples of application	Observation and Comment
ABAQUS	<p>Numerical investigation on the creep damage induced by void growth in heat affected zone of weldments [10]</p> <p>Benchmarks for finite element analysis of creep continuum damage mechanics [4]</p>	<p>The developer must develop a user-subroutine in junction with ABAQUS commercial FE code such as ABAQUS - UMAT damage code to analysis the creep CDM numerical problem [4].</p> <p>It can access to a wide range of element types, material models and other facilities such as efficient equation solvers, not normally available in in-house FE codes.</p> <p>It does not currently permit the removal of failed elements from the boundary-value problem during the solution process [11]</p>
ANSYS	<p>Development of a creep-damage model for non-isothermal long-term strength analysis of high-temperature components operating in a wide stress range [12]</p> <p>Numerical benchmarks for creep-damage</p>	<p>ANSYS uses full Newton-Raphson scheme for global solution to achieve better convergence rate [13]. The material matrix must be consistent with the material constitutive integration scheme for the better convergence rate of the overall Newton-Raphson scheme.</p> <p>To ensure the overall numerical stability, the user should ensure that the</p>

	modelling [13]	integration scheme implemented in subroutine is stable. Developing the user-subroutine for analyzing creep damage problems is very complex and inefficient [14].
MSC.Marc software	Numerical modelling of GFRP laminates with MSC.Marc system and experimental validation [15]	Marc is a powerful, general-purpose, nonlinear finite element analysis solution to accurately simulate the response of your products under static, dynamic and multi-physics loading scenarios. Developing the user-subroutine for analyzing creep damage problems is very complex and inefficient [14]
RFPA2D-Creep	Research on the closure and creep mechanism of circular tunnels[16]	RFPA2D-Creep introduces the degeneration equation on the mechanical characteristics of micro-element based on the meso-damage model in order to reveal the relationship between the damage accumulation, deformation localization and integral accelerated creep. The failed element cannot be removed and the accuracy should be improved [11]

### 2.2 The in-house finite element software

The in-house finite element software developed and used for creep damage analysis are listed and commented.

**Table 2 The main in-house finite element software.**

FE software & author	Characterization	Observation and Comment
FE-DAMAGE T.H. Hyde et al	<p>FE-DAMAGE is written in FORTRAN and developed at University of Nottingham [4]. Facilities for creep continuum damage analysis are included in which a single damage parameter constitutive equation is adopted.</p> <p>The failed elements from the boundary-value problem</p>	<p>The Object Oriented Programming (OOP) approach is not used in programming this software [4].</p> <p>The Object Oriented Programming (OOP) approach</p>

	can be removed during the solution process	could be used in future.
DAMAGE XX D.R. Hayhurst, et al	DAMAGE XX is 2-D CDM-based FE solver, which has been developed over three decades by a number of researchers [17].  The failed elements from the boundary-value problem can be removed during the solution process. The running speed of the computer code has been increased by vectorization and parallel processing on the Cray X-MP/416	The inability to solve problems with large numbers of FEs and degrees of freedom [18]. The speed of the numerical equation solver, which occupies a major part of the computer resource required. Fourth order integration scheme was used the detailed published might be in correct according to Ling et al [6].
DAMAGE XXX R.J. Hayhurst M.T. Wang	DAMAGE XXX is developed to model high-temperature creep damage initiation, evolution and crack growth in 3-D engineering components [19].  The failed elements from the boundary-value problem can be removed during the solution process  It is running on parallel computer parallel architectures. The tetrahedral elements are used in the DAMAGE XXX [17].	The speed of the numerical equation solver, which occupies a major part of the computer resource required. Fourth order integration scheme was used the detailed published might be in correct according to Ling et al [6].

DNA (Damage Non-linear Analysis)  G.Z. Voyiadjis, et al	DNA stands for Damage Nonlinear Analysis. It was at Louisiana State University in Baton Rouge. It includes the elastic, plastic and creep damage analysis of materials incorporating damage effects [20]. Both linear and nonlinear analysis options are available in DNA.  The failed elements from the boundary-value problem cannot be removed during the solution process	Number of nodes in a problem must not exceed 3000, the number of elements in a problem must not exceed 400 [20].  It is a 32-bit DOS executable file which can only run under the Windows 96/98/NT operating system.
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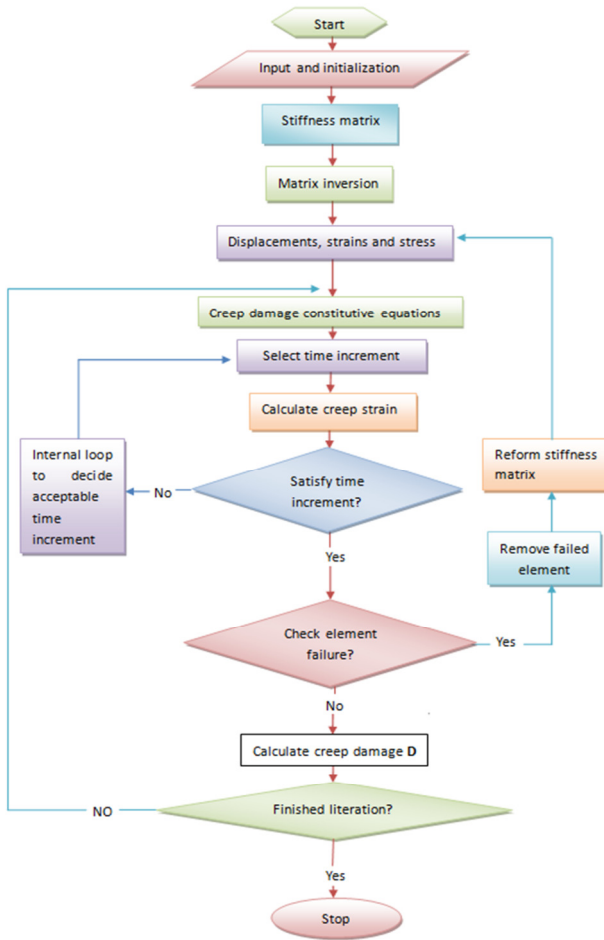
2.3 Why the in-house computational software.

FE standard packages can only obtain the capability for creep damage analysis by developing material user subroutine for investigating creep damage problems, which is very complex and not accurate [21]. Computational capability such as Continuum Damage Mechanics (CDM) for creep damage analysis is not readily available in the industrial standard FE packages. FE standard packages such as ABAQUS does not currently permit the failure of and the removal of the failed element from the boundary-value problems during the solution process [11]. Thus, there still have advantages in developing and using in-house finite element software.

**3. The Development of the New Finite Element Software**

3.1 The general structure of the finite element software

The structure of developing in-house finite element software for creep damage analysis is listed in Figure 1.



**Fig. 1** The structure of developing new FE software.

The steps for the development of finite element software can be summarized in:

1. Input the definition of a specific FE model including nodes, element, material property, boundary condition, as well as the computational control parameters
2. Calculate the initial elastic stress and strain
3. Integrate the constitutive equation and update the field variables such as creep strain, damage, stress; the time step is controlled
4. Remove the failed element [17] and update the stiffness matrix
5. Stop execution and output results

### 3.2 The equilibrium equations

Assume that the total strain  $\varepsilon$  can be partitioned into

the elastic and creep strains, thus the total strain increment can be expressed as:

$$\Delta \varepsilon = \Delta \varepsilon^e + \Delta \varepsilon^c \quad (1)$$

Where  $\Delta \varepsilon$ ,  $\Delta \varepsilon^e$  and  $\Delta \varepsilon^c$  are increments in total, elastic and creep strain components, respectively [22].

The stress increment is related to the elastic and creep strain increments by:

$$\Delta \sigma = \mathbf{D} (\Delta \varepsilon - \Delta \varepsilon^c) \quad (2)$$

Where  $\mathbf{D}$  is the stress-strain matrix and it contains the elastic constants.

The stress increments are related to the incremental displacement vector  $\Delta u$  by:

$$\Delta \sigma = \mathbf{D} (\mathbf{B} \Delta u - \Delta \varepsilon^c) \quad (3)$$

Where  $\mathbf{B}$  is strain matrix. The equilibrium equation to be satisfied any time can be expressed by:

$$\int v \mathbf{B}^T \Delta \sigma dv = \Delta \mathbf{R} \quad (4)$$

Where  $\Delta \mathbf{R}$  is the vector of the equivalent nodal mechanical load and  $v$  is the element volume. Combining equations (3) and (4):

$$\int v \mathbf{B}^T \mathbf{D} (\mathbf{B} \Delta u - \Delta \varepsilon^c) dv = \Delta \mathbf{R} \quad (5)$$

### 3.3 Sample creep damage constitutive equations

The creep damage constitutive equations are proposed to depict the behaviors of material during creep damage (deformation and rupture) process, especially for predicting the lifetime of material. One example is Kachanov-Rabatnov-Hayhurst (KRH) constitutive equations which is popular and is introduced here [23].

- Uni-axial form

$$\dot{\varepsilon} = A \sinh\left(\frac{B\sigma(1-H)}{(1-\varphi)(1-\omega)}\right) \quad (6.1)$$

$$\dot{H} = \frac{h}{\sigma} \left(1 - \frac{H}{H^*}\right) \dot{\varepsilon} \quad (6.2) \quad (6)$$

$$\dot{\varphi} = \frac{Kc}{3} (1 - \varphi)^4 \quad (6.3)$$

$$\dot{\omega} = C \dot{\varepsilon}^* \quad (6.4)$$

Where  $A$ ,  $B$ ,  $C$ ,  $h$ ,  $H^*$  and  $Kc$  are material parameters.  $H$  ( $0 < H < H^*$ ) indicates strain hardening during primary creep,  $\varphi$  ( $0 < \varphi < 1$ ) describes the evolution of spacing of the carbide precipitates [23].

- Multi-axial form

$$\begin{cases} \dot{\epsilon}_{ij} = \frac{3S_{ij}}{2\sigma_e} \text{Asinh}\left(\frac{B\sigma_e(1-H)}{(1-\varphi)(1-\omega)}\right) & (7.1) \\ \dot{H} = \frac{h}{\sigma_e} \left(1 - \frac{H}{H^*}\right) \dot{\epsilon} & (7.2) \\ \dot{\varphi} = \frac{K_c}{3} (1 - \varphi)^4 & (7.3) \\ \dot{\omega} = C \dot{\epsilon}_e \left(\frac{\sigma_1}{\sigma_e}\right)^v & (7.4) \end{cases} \quad (7)$$

Where  $\sigma_e$  is the Von Mises stress,  $\sigma_1$  is the maximum principal stress and  $v$  is the stress state index defining the multi-axial stress rupture criterion [23].

The intergranular cavitation damage varies from zero, for the material in the virgin state, to 1/3, when all of the grain boundaries normal to the applied stress have completely cavitated, at which time the material is considered to have failed [24]. Thus, the critical value of creep damage is set to 0.3333333 in the current program. Once the creep damage reaches the critical value, the program will stop execution and the results will be output automatically. Other type of creep damage constitutive equations will be incorporated in the FE software in future.

### 3.4 The integration scheme

The FEA solution critically depends on the selection of the size of time steps associated with an appropriate integration method. Some integration methods have been reviewed in previous work [3]. In the current version, Euler forward integration subroutine, developed by colleagues [25], was adopted here for simplicity.

$$\begin{cases} \epsilon_{n+1} = \epsilon_n + \dot{\epsilon} * \Delta t & (8.1) \\ H_{n+1} = H_n + \dot{H} * \Delta t & (8.2) \\ \varphi_{n+1} = \varphi_n + \dot{\varphi} * \Delta t & (8.3) \\ \omega_{n+1} = \omega_n + \dot{\omega} * \Delta t & (8.4) \\ t_{n+1} = t_n + \Delta t & (8.5) \end{cases} \quad (8)$$

It is noted that D02BHF (NAG) [26] integrates a system of first-order ordinary differential equations solution using Runge-Kutta-Merson method. This subroutine can be adopted in the FEA software of creep damage analysis development, and a detailed instruction on how to use it was published by the

company [26]. A more sophisticated Runge-Kutta type integration scheme will be adopted and explored in future.

### 3.5 The finite element algorithm for updating stress

The Absolute Method [27] has been given for the solution of the structural creep damage problems. The principle of virtual work applied to the boundary value problem is given:

$$P_{load} = [K_v] * TOTD - P_c \quad (9)$$

Where  $P_{load}$  is applied force vector, and  $[K_v]$  is the global stiffness matrix, which is assembled by the element stiffness matrices  $[K_m]$ ; TOTD is the global vector of the nodal displacements and  $P_c$  is the global creep force vector.

$$[K_m] = \int [B]^T [D] [B] dx dy \quad (10)$$

The  $[B]$  and  $[D]$  represent the strain-displacement and stress-strain matrices respectively.

$$TOTD = [K_v]^{-1} * (P_{load} + P_c) \quad (11)$$

The initial  $P_c$  is zero and the Choleski Method [27] is used for the inverse of the global stiffness matrix  $[K_v]$ . By giving the  $P_{load}$ , the elastic strain  $\epsilon_{ek}$  and the elastic stress  $\sigma_{ek}$  for each element can be obtained:

$$\epsilon_{ek} = [B] * ELD \quad (12)$$

$$\sigma_{ek} = [D] * \epsilon \quad (13)$$

The element node displacement ELD can be found from the global displacement vector and the creep strain rate  $\epsilon_{ckrate}$  for each element can be obtained by substituting the element elastic stress into the creep damage constitutive equations. The creep strain can be calculated as:

$$\epsilon_{ck}(t + \Delta t) = \epsilon_{ck}(t) + \epsilon_{ckrate} * \Delta t \quad (14)$$

The node creep force vectors for each element are given by:

$$P_{ck} = [B]^T [D] * \epsilon_{ck} \quad (15)$$

The node creep force vector  $P_{ck}$  can be assembled into the global creep force vector  $P_c$  and the  $P_c$  is used to up-date equation (9). Thus, the elastic strain can be updated:

$$\epsilon_{totk} = [B] * ELD = \epsilon_{ek} + \epsilon_{ck} \quad (16)$$

$$\epsilon_{ek} = [B] * ELD - \epsilon_{ck} \quad (17)$$

Where the  $\epsilon_{totk}$  and  $\epsilon_{ck}$  represent the total strain and creep strain for each element respectively; and the elastic strain  $\epsilon_{ek}$  is used to up-date the equation (13).

3.6 The standard FE library and subroutines

In the development of this software, the existing FE library and subroutines such as [27] were used in programming. The subroutines can perform the tasks of finite element meshing, assemble element matrices into system matrices and carry out appropriate equilibrium, eigenvalue or propagation calculations. Some subroutines used in programming are reviewed in Table 3.

Table 3 The existing FE library and subroutines.

The standard subroutine	Function
Subroutine geometry_3tx	To form the coordinates and node vector for a rectangular mesh of uniform 3-node triangles
Subroutine formkb and Subroutine formkv	To assemble the individual element matrices to form the global matrices
Subroutine sparin and Subroutine spabac	To solve the sets of linear algebraic equations based on the Cholesky direct solution method

4. Preliminary Validation and Discussion

4.1 The validation of plane stress problem

The validation of new software for plane stress was performed and it was conducted via a two-dimensional tension model. The length of a side is set to 1 meter. The Young's modulus E and Poisson's ratio  $\nu$  are set to 1000MPa and 0.3 respectively. A uniformly distributed linear load 40KN/m was applied on the top line of this uni-axial tension model.

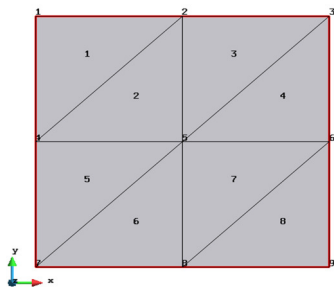


Fig. 2 2D plane stress tension mode.

The theoretical stress in Y direction is 40KN/m<sup>2</sup>.

The stress in X direction should be zero. These stress values should remain the same throughout the creep test up to failure.

Samples of the stress obtained from FE software with the stress updating invoked due to creep deformation are shown in Figure 3 and Figure 4.

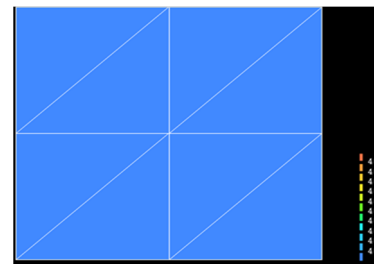


Fig. 3 The stress distribution in Y direction at rupture time.

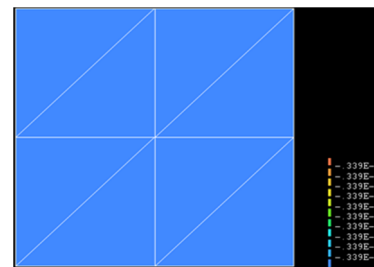


Fig. 4 The stress distribution in X direction at rupture time.

Using the theoretical stress value into the uni-axial version of creep constitutive equations and a 0.1 hour time step with Euler forward integration method, the theoretical rupture time, creep strain rate, creep strain and damage can be obtained by a excel program [28] and some of them are shown in Table 4.

Table 4 The results obtained from excel program

Rupture time	Creep strain	Creep damage
104062	0.179934333	0.33333335

Using the uni-axial version of creep constitutive equations and a 0.1 hour time step with Euler forward

integration method, the rupture time, creep strain rate, creep strain and damage obtained from FE software at failure were obtained and are shown in Table 5.

**Table 5 The results obtained from FE software for plane stress problem**

Element number	Rupture time	Strain rate	Creep strain	Creep damage
Element No.1	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00
Element No.2	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00
Element No.3	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00
Element No.4	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00
Element No.5	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00
Element No.6	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00
Element No.7	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00
Element No.8	0.1040E+06	0.6540E-04	0.1798E+00	0.3334E+00

The percentage errors of FE results against theoretical results are shown in Table 6.

**Table 6 The percentage errors**

$\text{Rupture time percentage error} = \left  \frac{104000 - 104062}{104062} \right  \times 100 = 0.0596\%$
$\text{Creep strain percentage error} = \left  \frac{0.1798 - 0.179934333}{0.179934333} \right  \times 100 = 0.0747\%$
$\text{Damage percentage error} = \left  \frac{0.3334 - 0.33333335}{0.33333335} \right  \times 100 = 0.02\%$

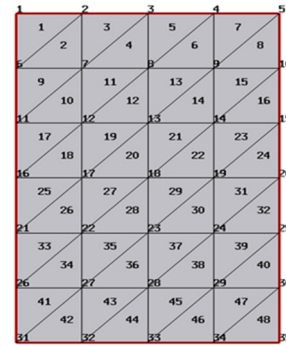
A comparison of the results shown in Table 4 and Table 5 and an examination of the percentage errors shown in Table 6 clearly show the results obtained from the FE software do agree with the expected

theoretical values and the percentage errors are negligible.

In the current version, Euler forward integration subroutine, developed by colleagues [25] was adopted here. Rupture time, strain rate, creep strain and damage obtained from FE software have been revealed that the FE results have a good agreement with the theoretical values.

#### 4.2 The validation of plane strain problem

The validation of this software for plane stress was performed and it was conducted via a uni-axial tension model. The width of this model is set to 5 meters. The Young's modulus  $E$  and Poisson's ratio  $\nu$  are set to 1000 MPa and 0.3 respectively. A uniformly linear distributed load 10 KN/m was applied on the top line of this model.



**Fig. 5 Plane strain tension model.**

The theoretical stress in Y direction can be shown as:

$$\sigma_y = \frac{P}{A} = \frac{50}{5.0} = 10 \text{ KN/m}^2$$

The theoretical stress in Z direction can be shown as:

$$\sigma_z = E * \epsilon_z = E * \nu * \epsilon_y = E * \nu * \frac{\sigma_y}{E} = 3 \text{ KN/m}^2$$

The stress and displacement obtained from FE software with the stress updating invoked due to creep deformation are shown in Figure 6 and Figure 7. The displacements in x and y direction is shown by Figure 8 and Figure 9, respectively.



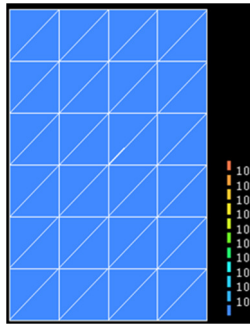


Fig. 6 Stress distribution in Y direction at rupture time.

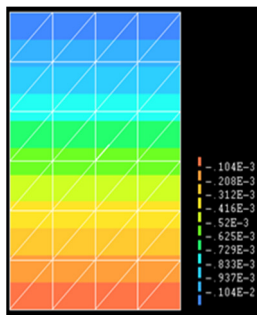


Fig. 7 Stress distribution in Z direction at rupture time.

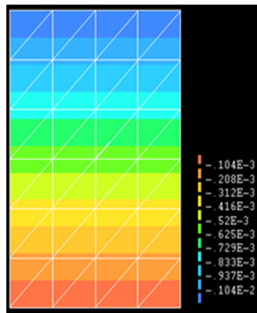


Fig. 8 Displacement distribution in Y axis at rupture time.

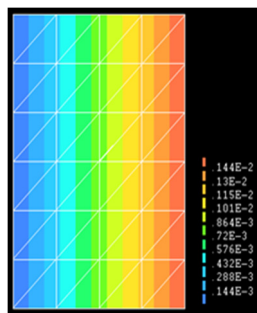


Fig. 9 Displacement distribution in X axis at rupture time.

Using the theoretical stress value into the

multi-axial version of creep constitutive equations, the theoretical rupture time and damage can be obtained without stress update by a testified subroutine [25] and the results are shown in Table 7.

Table 7 The theoretical rupture time and creep damage for plane strain case

Rupture time	Creep damage
180460	0.3333334

The damage obtained from FE software at failure.

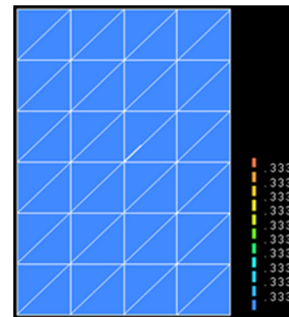


Fig. 10 The damage distribution on 180230h.

The Figure 6 and Figure 7 show the results obtained from the FE software do agree with the expected theoretical values.

The displacement is distributed reasonable in Figure 8 and Figure 9. Table 7 and Figure 10 have revealed that rupture time and damage obtained from FE software have a good agreement with the theoretical values obtained from the subroutine [29].

#### 4.3 The validation of axisymmetric problem

The validation of new software for the axisymmetric problem was performed and it was conducted via a uni-axial tension model. A uniformly distributed tensile force 0.375 KN/m<sup>2</sup> was applied on the top line of this uni-axial tension model.

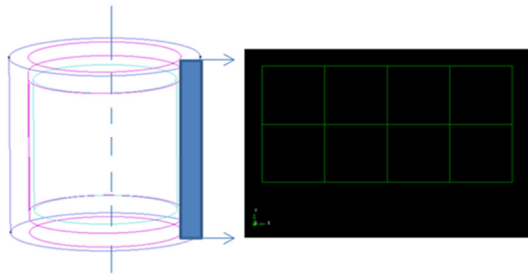


Fig. 11 The axisymmetric FE model.

Using the theoretical stress value into the multi-axial version of creep constitutive equations and a 0.1 hour time step with Euler forward integration method, the theoretical rupture time and damage can be obtained by a subroutine [29] and the results are shown in Table 8.

Table 8 The theoretical rupture time and creep damage for axisymmetric case

Rupture time	Creep damage
146080	0.3333334

The stress and displacement obtained from FE software with the stress updating invoked due to creep deformation are shown in Figure 12 and Figure 13.

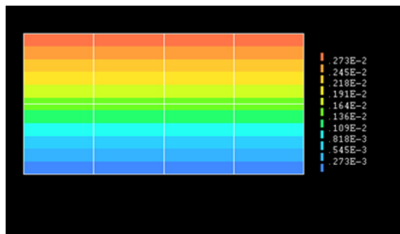


Fig. 12 Displacement distribution in Z axis.

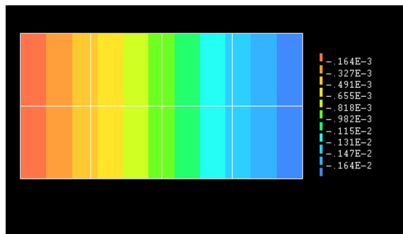


Fig. 13 The displacement distribution in r axis.

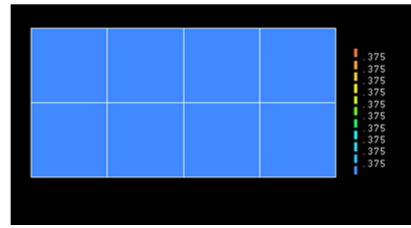


Fig. 14 Stress distribution in Z direction.

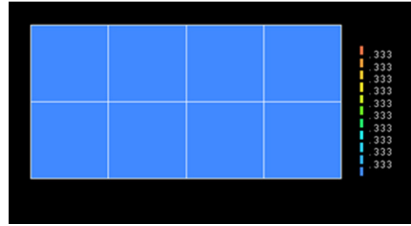


Fig. 15 Damage distribution on 143060h.

The stress has been uniformly distributed in Figure 14 and do agree with the theoretical values.

Other results are shown in Table 8 and Figure 15. Rupture time and damage obtained from FE software have been revealed that have a good agreement with the theoretical values obtained from the subroutine [29].

4.4 The validation of 3D problem

A preliminary validation of such software was performed and it was conducted via a three-dimensional uni-axial tension model given below. The length of a side is set to 1 meter and a uniformly distributed load 5KN was applied on the top surface of this uni-axial tension model.

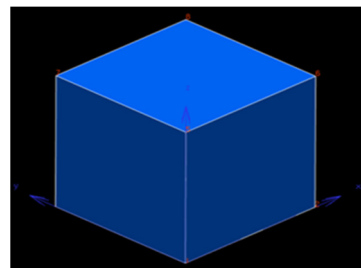


Fig. 16 The three-dimensional uni-axial tension model.

The theoretical stress in Z direction is 5KN/m<sup>2</sup>. The

stress in X and Y direction should be zero and these stress values should remain the same throughout the creep test up to failure. The stress obtained from FE software with the stress update program is shown in Table 9 at a 0.1 hour time step with Euler forward integration method.

**Table 9 The stress obtained from FE software with the stress update program**

<i>Integration point</i>	$\sigma_x$	$\sigma_y$	$\sigma_z$
No. 1	0.1545E-05	0.1377E-06	0.5000E+01
No. 2	0.4970E-06	0.7690E-06	0.5000E+01
No. 3	0.1068E-05	0.2017E-07	0.5000E+01
No. 4	0.5675E-06	0.1478E-06	0.5000E+01
No. 5	0.1760E-05	0.3392E-06	0.5000E+01
No. 6	0.1212E-05	0.8395E-06	0.5000E+01
No. 7	0.6717E-07	0.8630E-06	0.5000E+01
No. 8	0.6380E-06	0.1648E-06	0.5000E+01

Table 9 shows the results obtained from the FE software do agree with the expected theoretical values. The stress involving creep deformation and stress updating confirmed the uniform distribution of stresses, and the values of stress in Z direction obtained from FE software are correct, and the stress in X and Y direction is negligible.

The lifetime and creep strain at failure, and other field variable can be obtained for the simple tensile case illustrated above. They have been obtained by direct integration of the uni-axial version of constitutive equation for a given stress [30]. They have also been produced by the FE software. Table 10 is a summary and comparison of them.

**Table 10 The stress obtained from FE software with the stress update program**

<i>The results</i>	<i>Theoretical value</i>	<i>FE results</i>
Rupture time	98046	93540
Strain rate	0.000065438	0.000067522
Creep strain	0.179934333	0.182658312
Damage	0.33333337	0.33333334

Table 10 reveals that real values have a good agreement with the theoretical values obtained from the subroutine [30]. Work in this area is ongoing and will be reported in future.

## 5. Conclusion

This paper is to present the finite element method based on CDM to design FE software for creep damage mechanics. More specifically, it summarizes the current state of how to obtain such computational capability then it concludes with a preference of in-house software; secondly, it reports the development of such software including the development of finite element algorithms based on CDM for creep damage analysis, and a flow diagram of the structure of new finite element software has been completed to be guided in developing in-house FE software, and the use of some standard subroutines in programming; thirdly, the development and the validation of the finite element software conducted so far include plane stress, plane strain, axisymmetric case, and 3D case were reported.

Working in this area is ongoing and future development work includes: 1) development and incorporation the new constitutive equation subroutines; 2) intelligent and practical control of the time steps; 3) removal of failed element and update stiffness matrix; and 4) further validation. Further real case study to be conducted and reported.

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